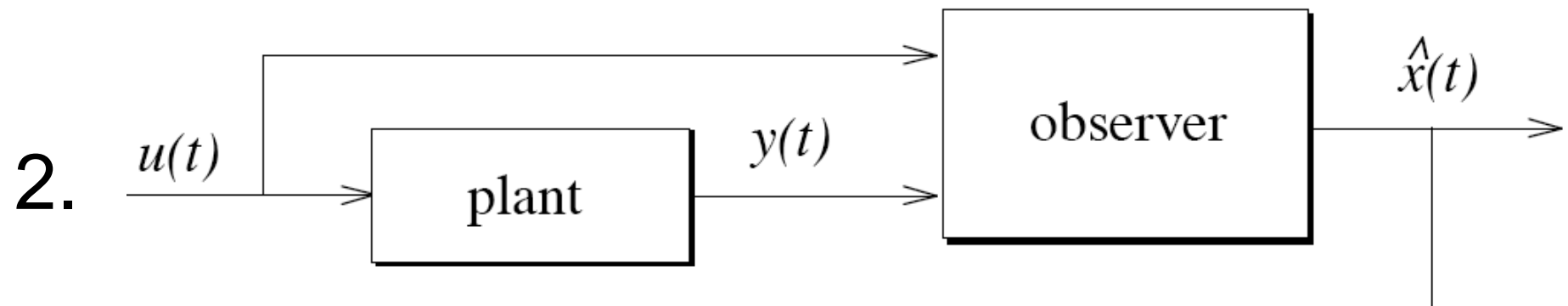


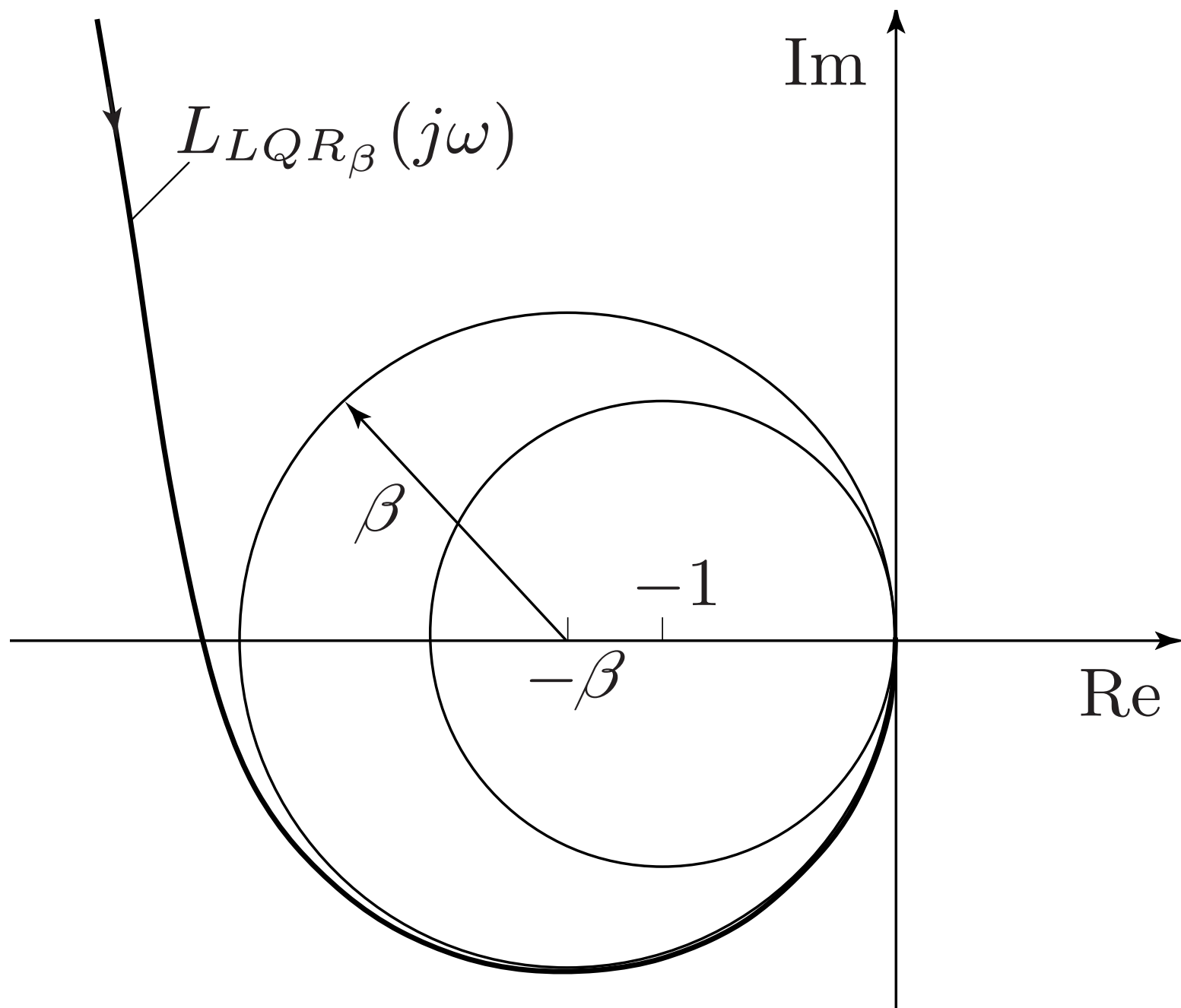
Plan:

1. $u(t) = f(x(t), t)$

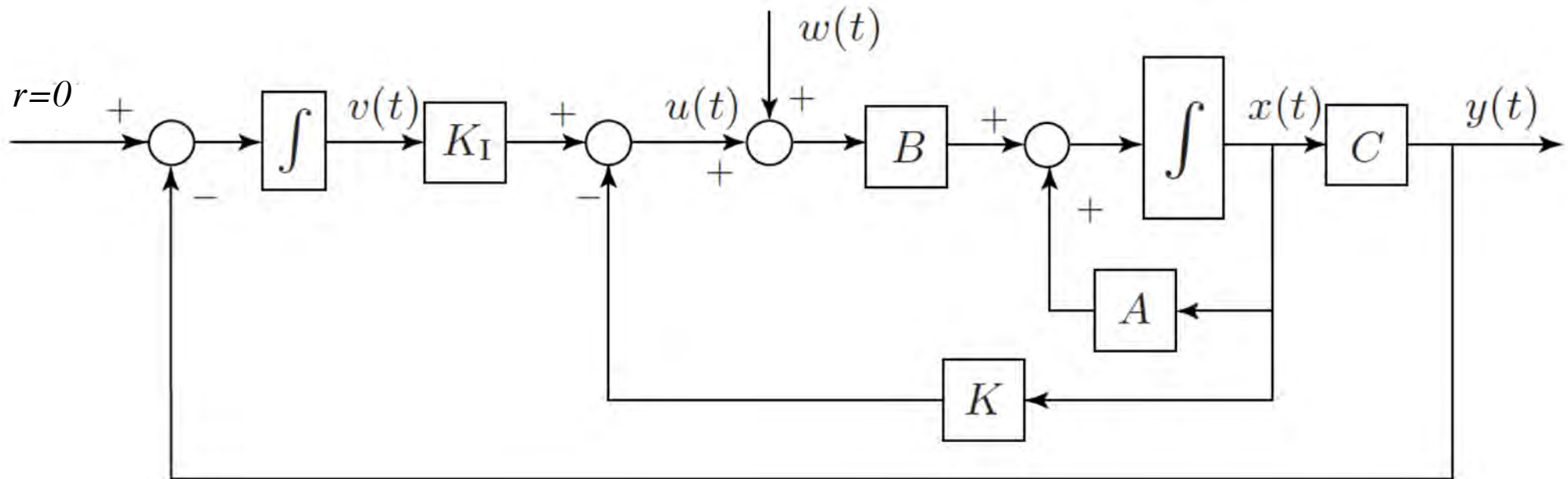


3.

4. Many improvements



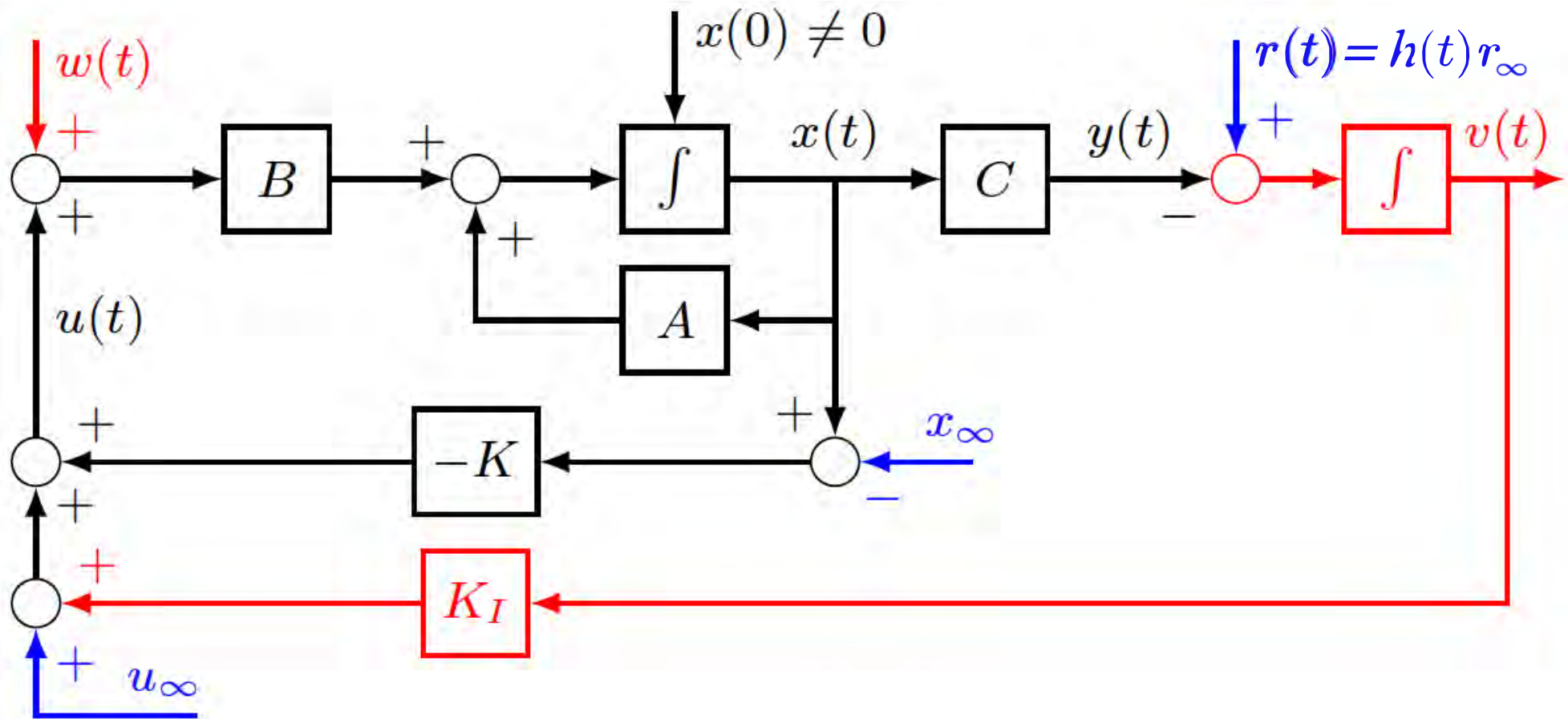
LQR-I Controller



Other structures possible

Re-arrange as standard LQR problem

Reference Tracking LQR-I



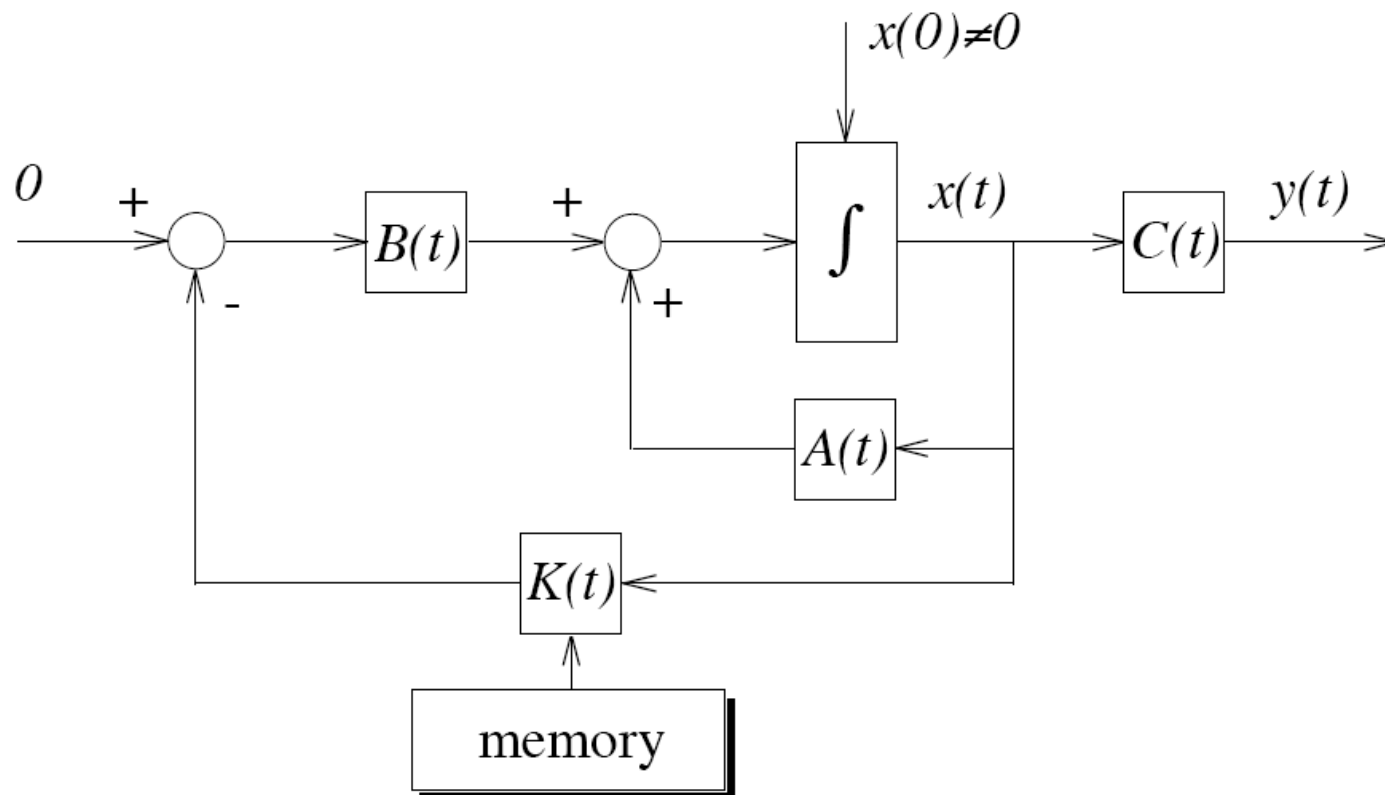
Full solution: See Section 4.4.3

Time-Varying Systems and Finite Horizons

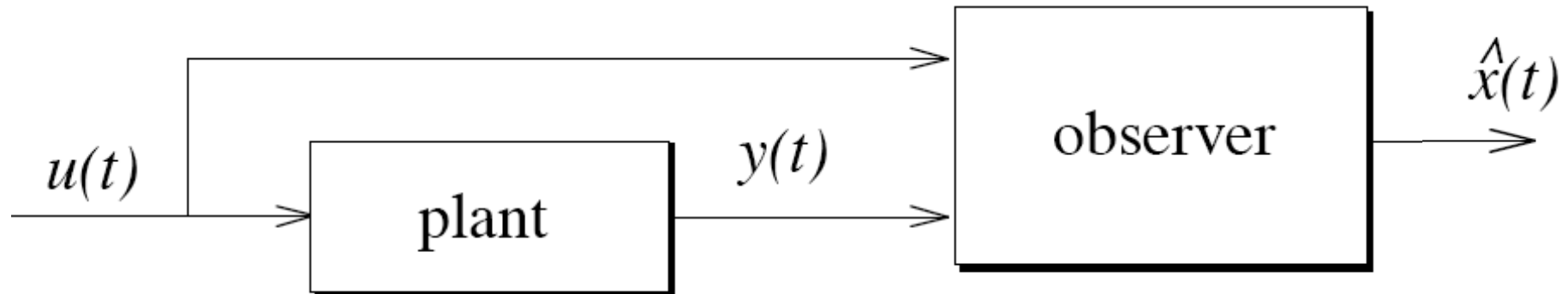
$$\frac{d}{dt}x(t) = A(t) \cdot x(t) + B(t) \cdot u(t), \quad x(t) \in \mathbb{R}^n, \quad u(t) \in \mathbb{R}^m, \quad x(t_a) = x_a$$

$$J(u) = x^T(t_b) \cdot P \cdot x(t_b) + \int_{t_a}^{t_b} (x^T(u(t)) \cdot Q(t) \cdot x(u(t)) + u^T(t) \cdot R(t) \cdot u(t)) dt$$

$$\frac{d}{dt}\Phi(t) = \Phi(t) \cdot B(t) \cdot R^{-1}(t) \cdot B^T(t) \cdot \Phi(t) - \Phi(t) \cdot A(t) - A^T(t) \cdot \Phi(t) - Q(t), \quad \Phi(t_b) = P$$

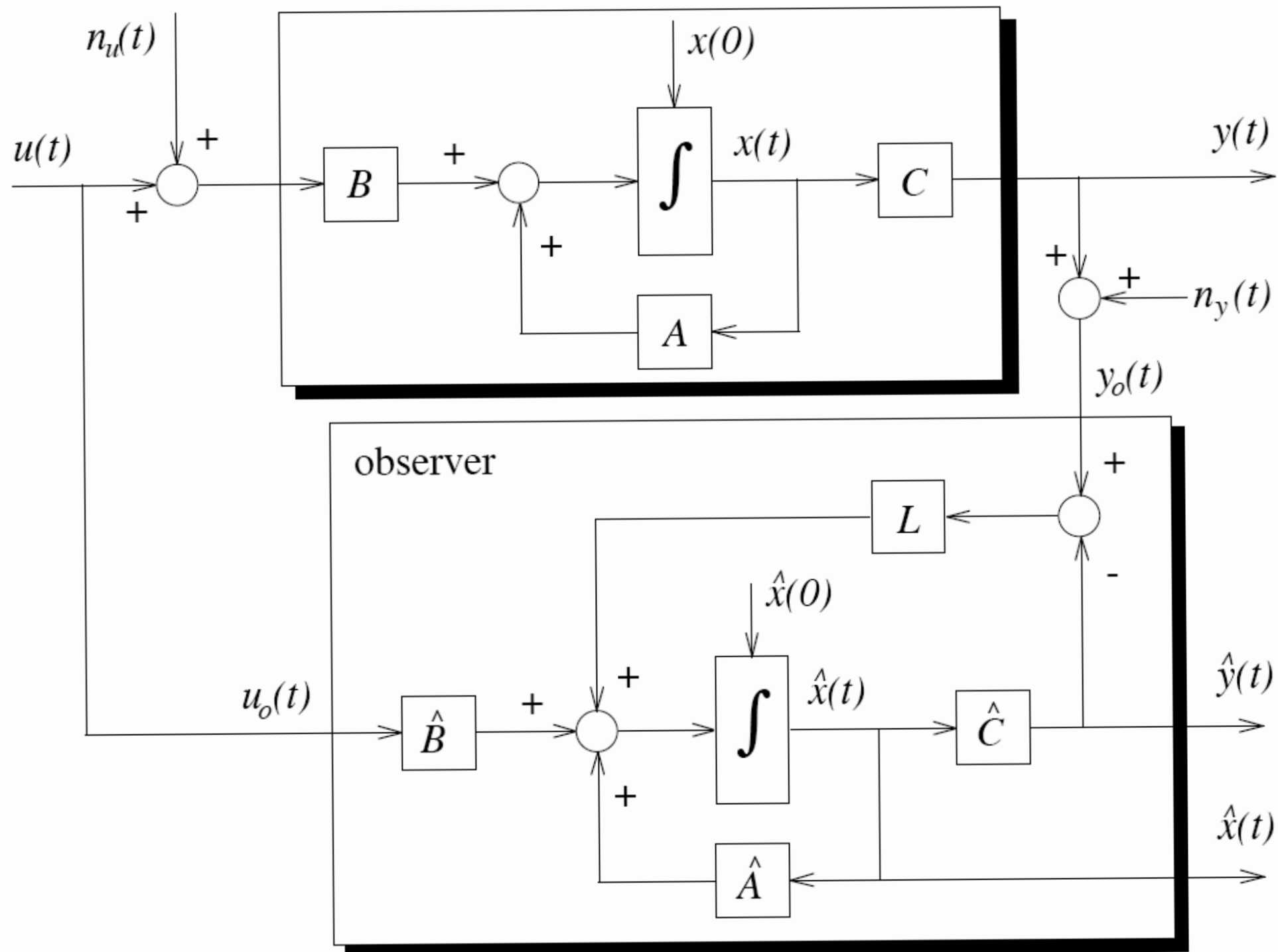


State Observers



Known: signals: $u(t)$ and $y(t)$
plant model $\{A, B, C, D\}$

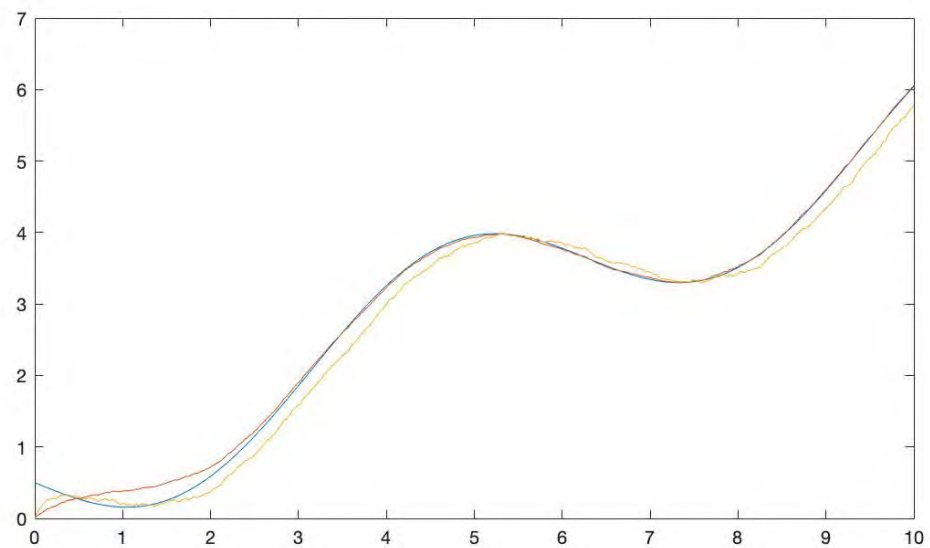
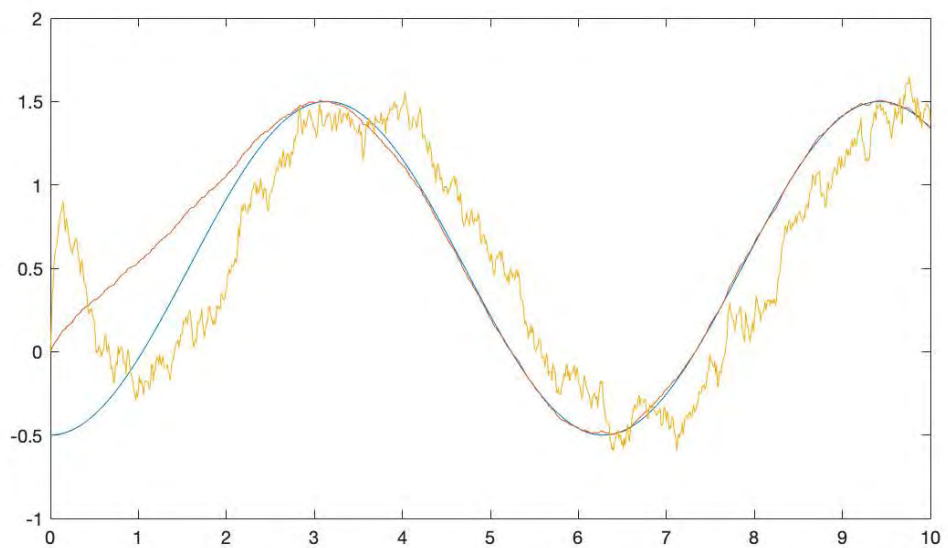
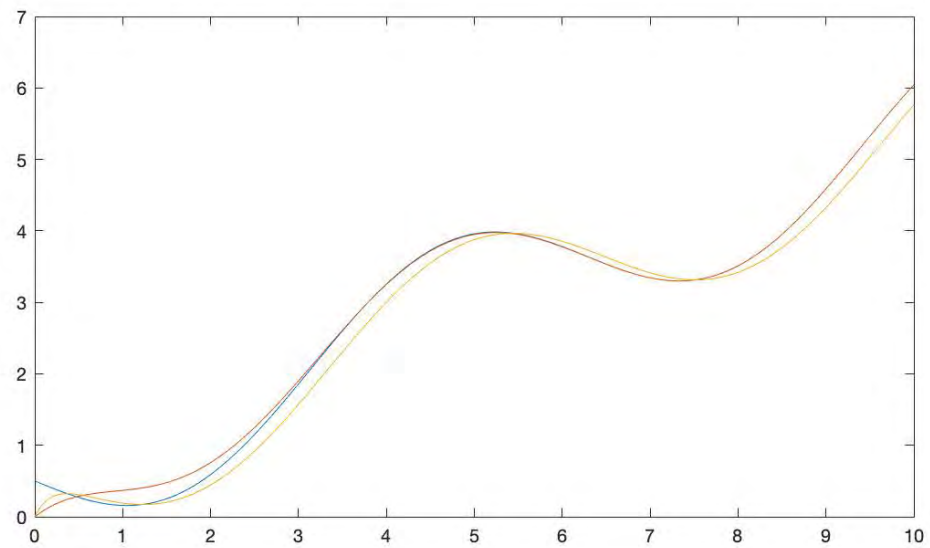
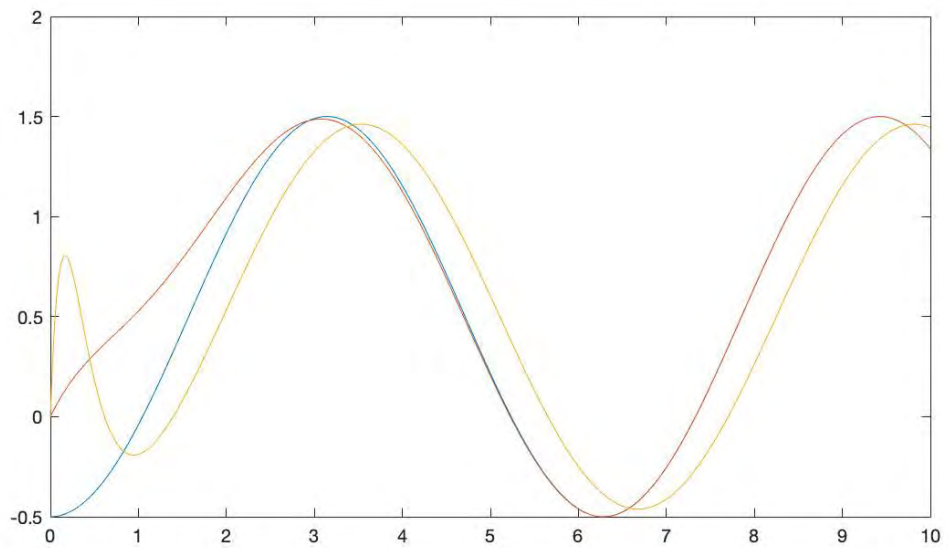
Find: Estimation $\hat{x}(t)$ of $x(t)$



$$\bar{x}(t) = x(t) - \hat{x}(t) \quad \frac{d}{dt}\bar{x}(t) = (A - L \cdot C) \cdot \bar{x}(t)$$

How to compute L ?

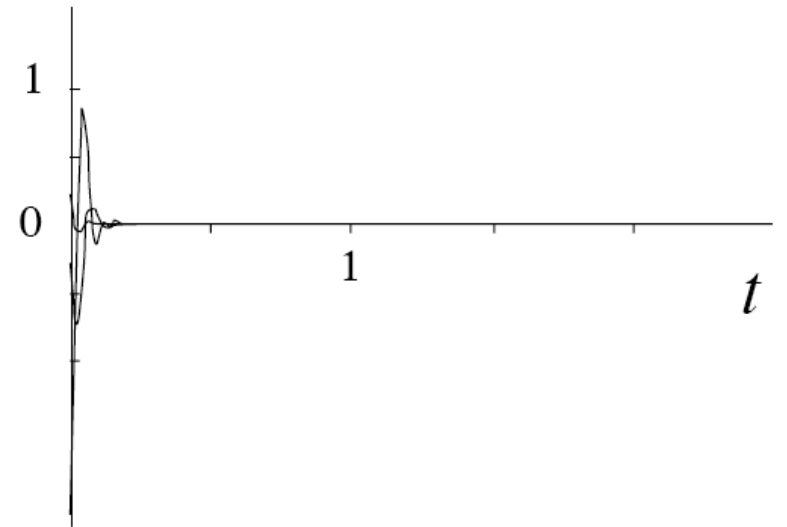
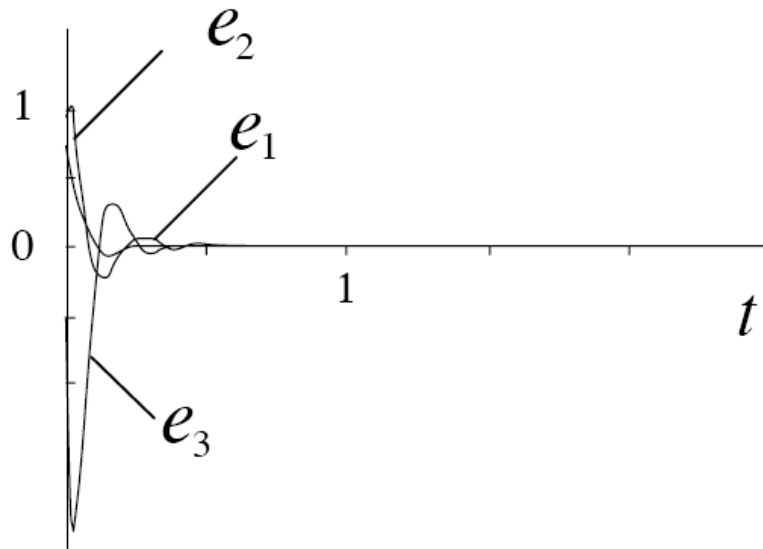
Dual LQR problem	A	\rightarrow	A^T
	B	\rightarrow	C^T
	$Q = \bar{C}^T \cdot \bar{C}$	\rightarrow	$\bar{B} \cdot \bar{B}^T$
	$R = r \cdot I$	\rightarrow	$q \cdot I$



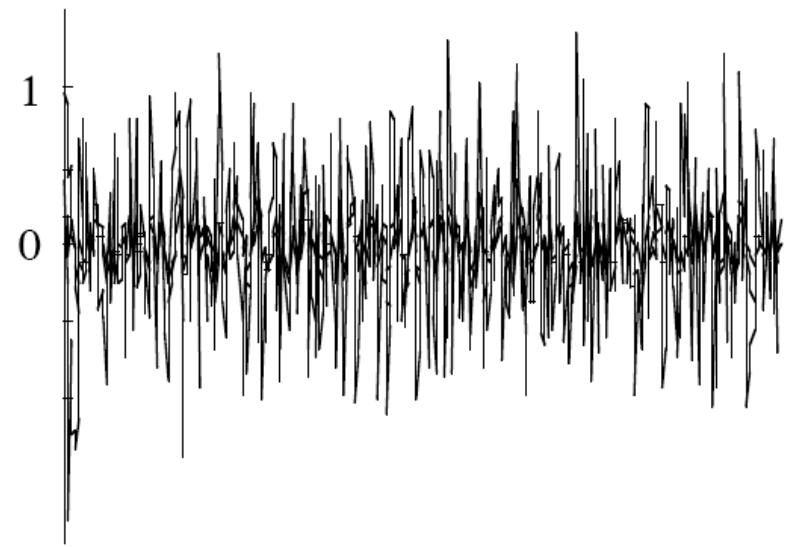
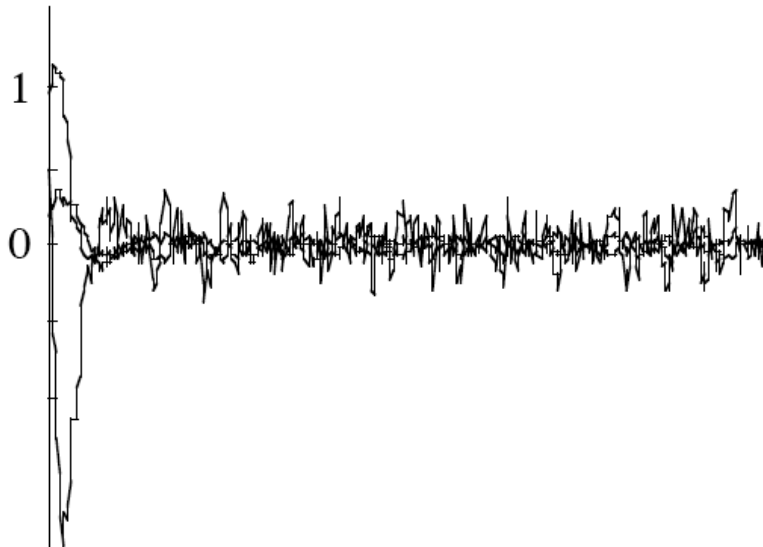
“slow” observer

“fast” observer

no
noise



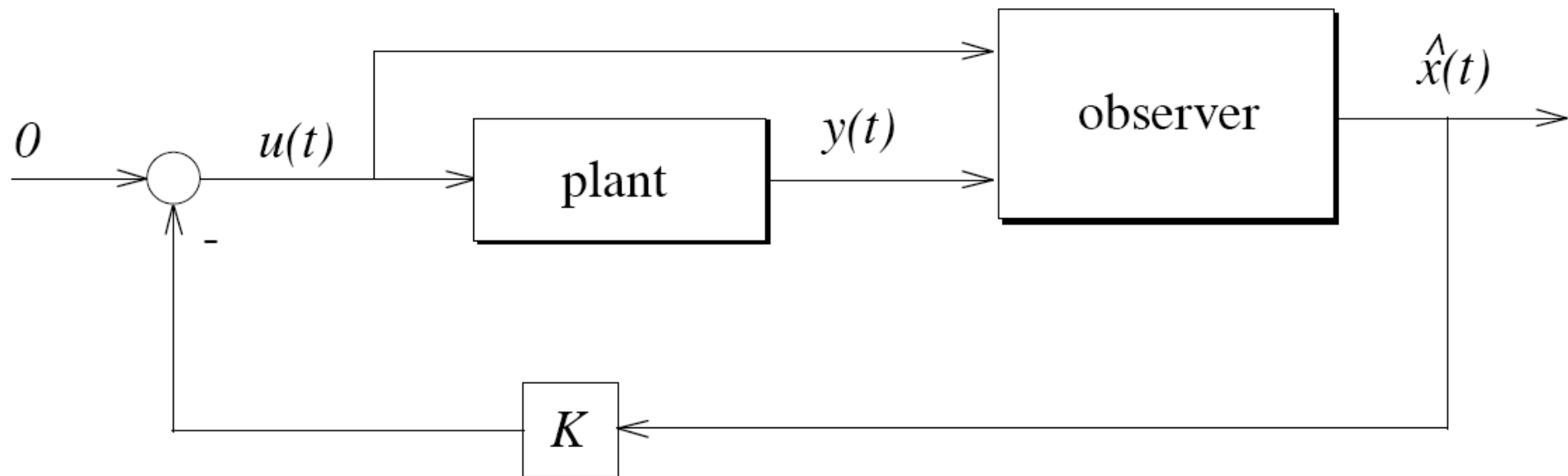
some
noise



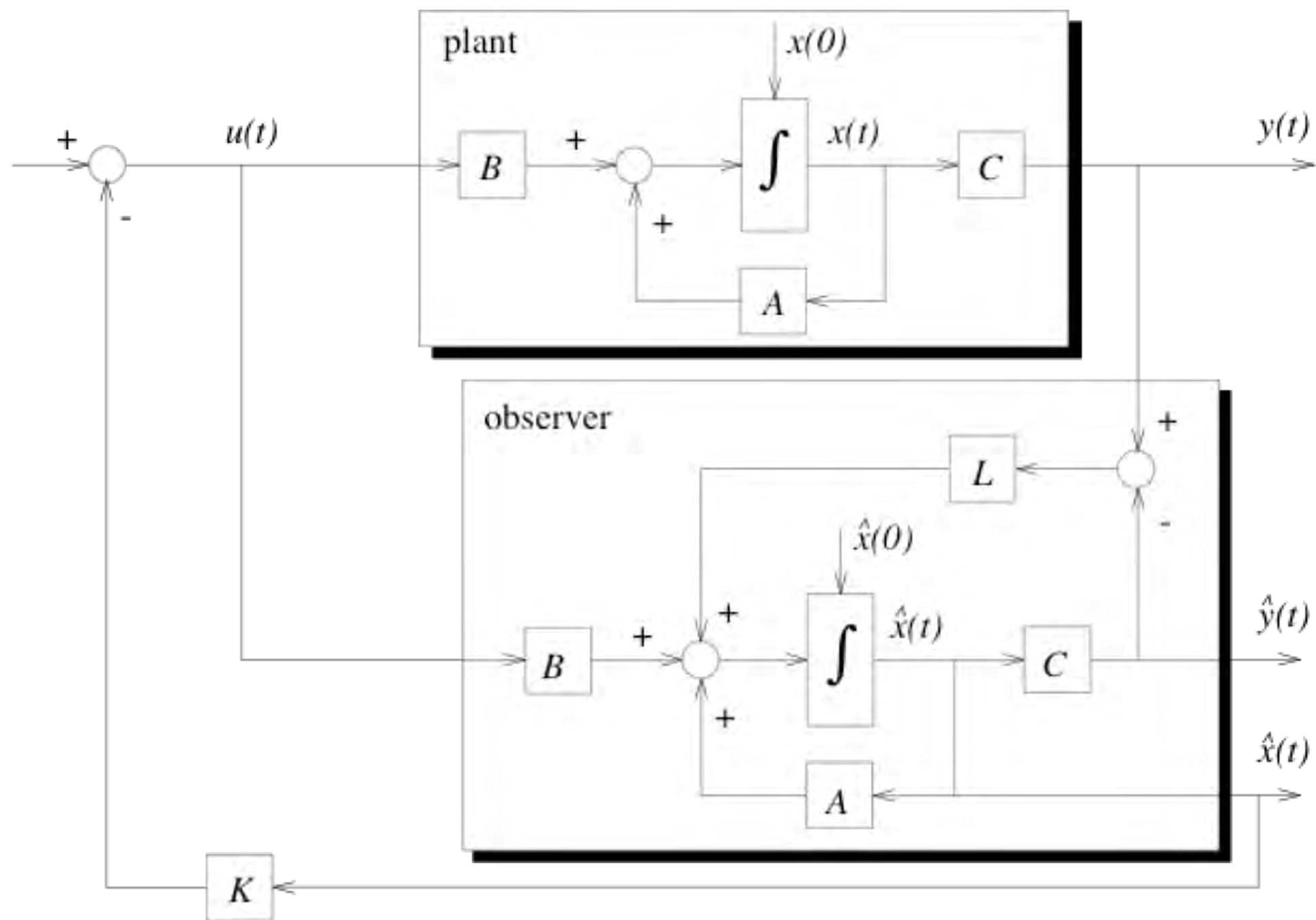
Lecture X – Observer-Based Output Feedback Control

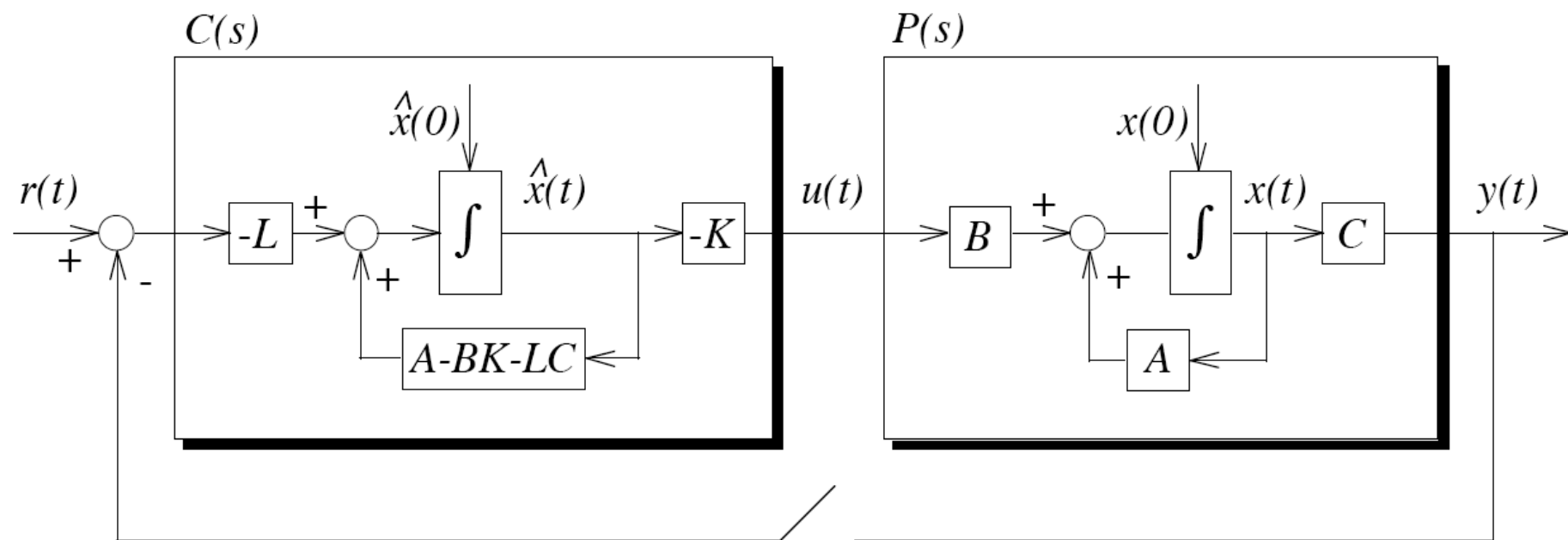
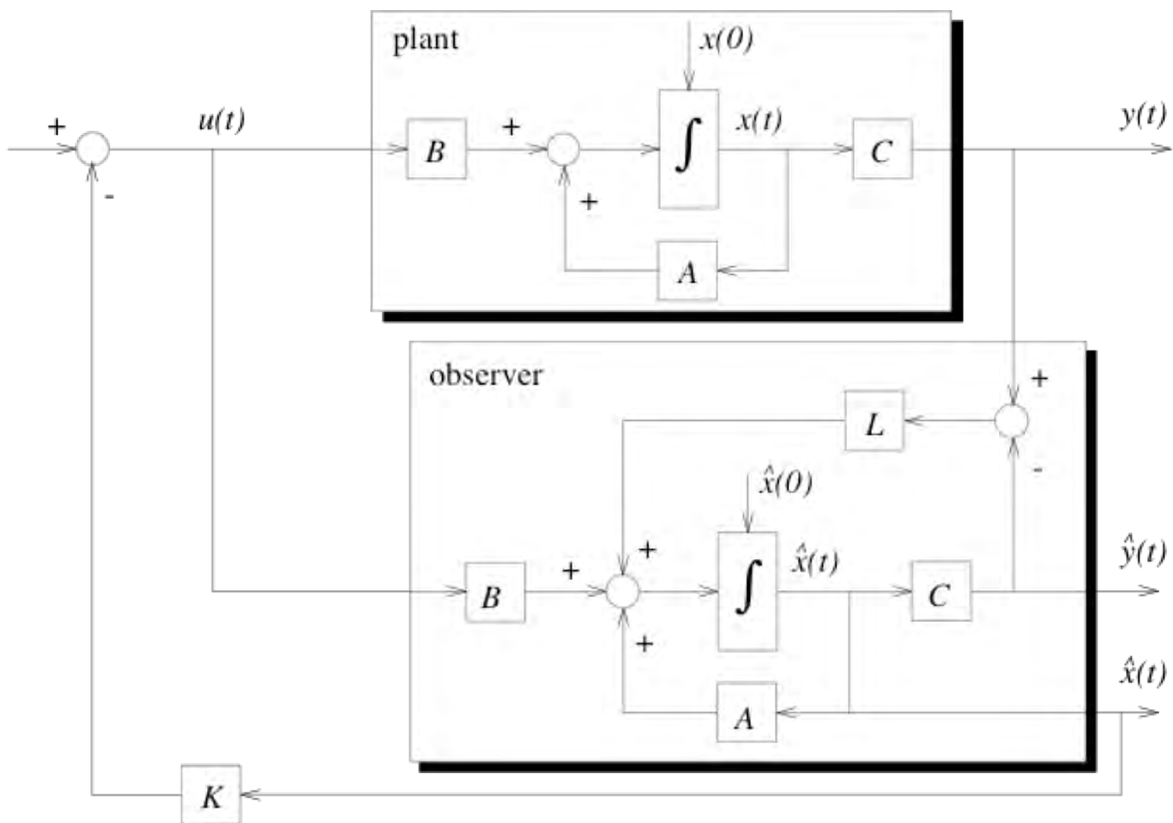
Output Feedback Controllers

$$u(t) = -K \cdot \hat{x}(t)$$



LQG(aussian): observer gain computed using dual LQR approach





LQG = Output-Feedback, i.e., realizable!

State vector

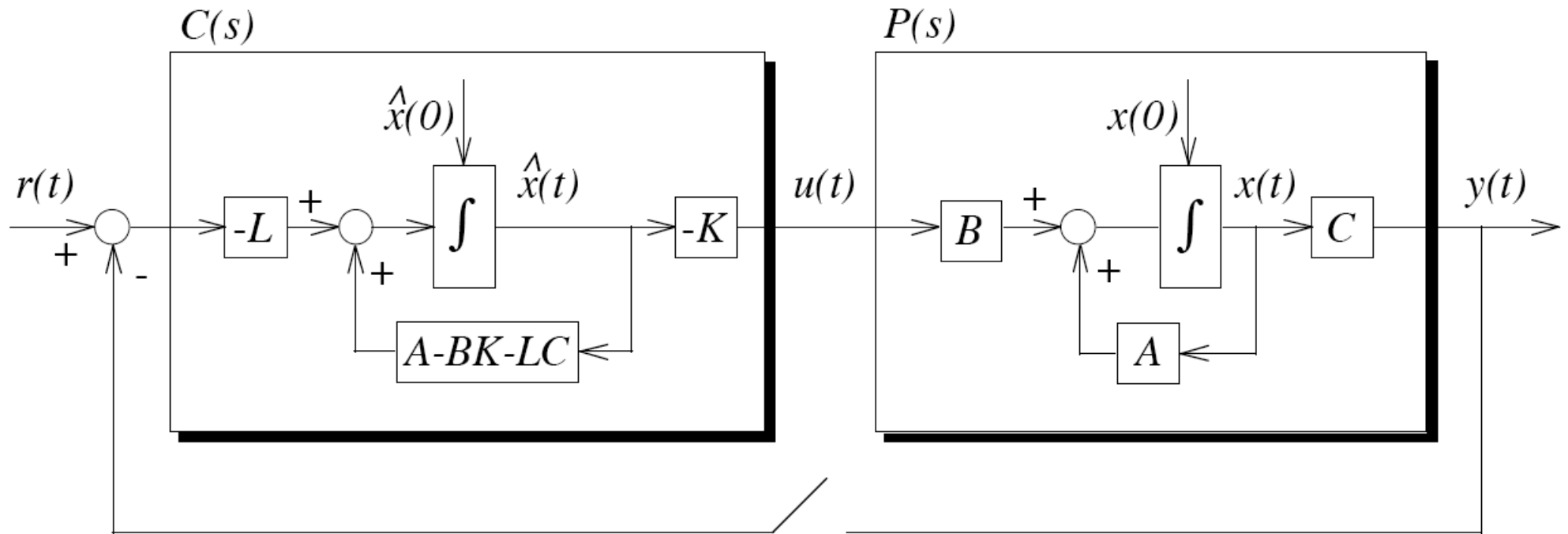
$$\tilde{x}(t) = \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix}$$

System description:

a) open loop \Rightarrow robustness

b) closed loop \Rightarrow stability

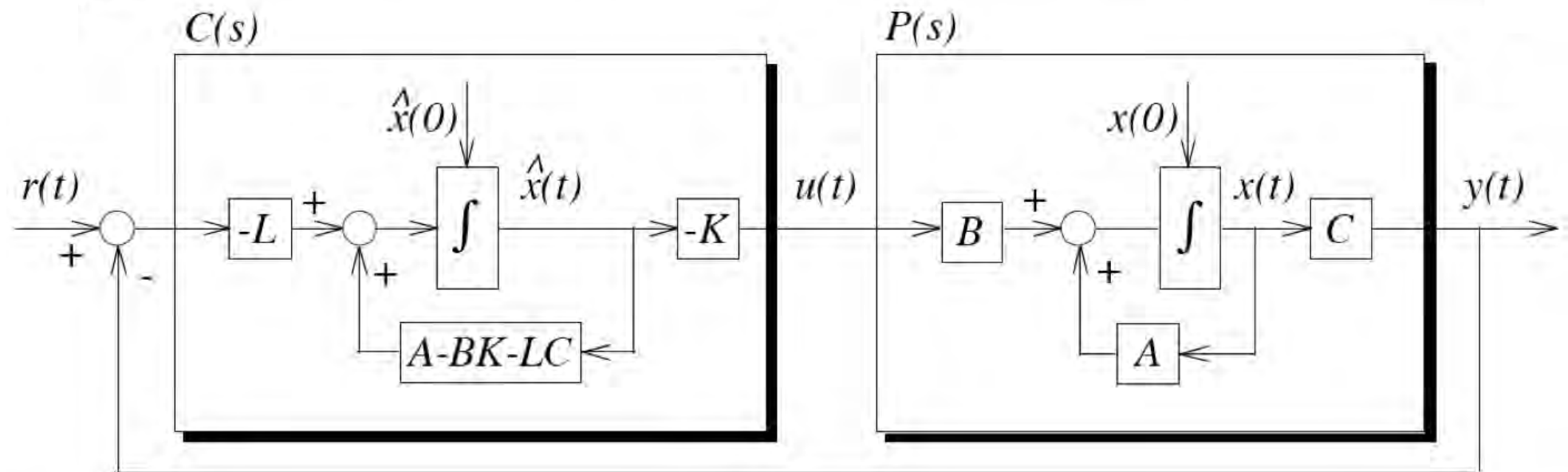
$$\frac{d}{dt}\tilde{x}(t) = \tilde{A}_{ol} \cdot \tilde{x}(t) + \tilde{B} \cdot r(t), \quad y(t) = \tilde{C} \cdot \tilde{x}(t)$$



$$\tilde{A}_{ol} = \begin{bmatrix} A & -B \cdot K \\ 0 & A - B \cdot K - L \cdot C \end{bmatrix}, \quad \tilde{B} = \begin{bmatrix} 0 \\ -L \end{bmatrix}, \quad \tilde{C} = \begin{bmatrix} C & 0 \end{bmatrix}$$

$$L_{LQG}(s) = C \cdot [sI - A]^{-1} \cdot B \cdot K \cdot [sI - (A - B \cdot K - L \cdot C)]^{-1} \cdot L$$

$$\frac{d}{dt}\tilde{x}(t) = \tilde{A}_{cl} \cdot \tilde{x}(t) + \tilde{B} \cdot r(t), \quad y(t) = \tilde{C} \cdot \tilde{x}(t)$$



$$\tilde{A}_{cl} = \begin{bmatrix} A & -B \cdot K \\ L \cdot C & A - B \cdot K - L \cdot C \end{bmatrix}$$

In particular, two points must be verified:

1. the stability of the closed-loop system (3.58); and
2. the robustness of the open-loop system (3.56).

Ad. 1.: Analyze \tilde{A}_{cl} , i.e., is this matrix Hurwitz
(all of its Eigenvalues negative real part)?
Result know as "Separation Principle", today.

Ad. 2: Analyze return difference $I+L_{LQG}(j\omega)$

The Separation Principle

Assume $A - B \cdot K$ and $A - L \cdot C$ are both Hurwitz matrices. Then the output-feedback closed-loop system is asymptotically stable and n of its eigenvalues coincide with the eigenvalues of $A - B \cdot K$ and its other n eigenvalues coincide with the eigenvalues of $A - L \cdot C$.

Proof: The closed-loop system can be compactly described using the $2n$ -dimensional vector

$$x_{cl}(t) = \begin{bmatrix} x(t) \\ \hat{x}(t) \end{bmatrix}$$

using the ODE

$$\frac{d}{dt}x_{cl}(t) = A_{cl} \cdot x_{cl}(t) + B_{cl} \cdot r(t), \quad y(t) = C_{cl} \cdot x_{cl}(t)$$

where the matrix A_{cl} , which fully determines the stability properties, is

$$\tilde{A}_{cl} = \begin{bmatrix} A & -B \cdot K \\ L \cdot C & A - B \cdot K - L \cdot C \end{bmatrix}$$

Using the coordinate transformation $x_{cl} = T \cdot z_{cl}$ with

$$T = \begin{bmatrix} I_{n \times n} & 0_{n \times n} \\ I_{n \times n} & -I_{n \times n} \end{bmatrix} = T^{-1}$$

yields the following transformed system equation

$$\frac{d}{dt} z_{cl}(t) = T^{-1} \cdot A_{cl} \cdot T \cdot z_{cl}(t) + T^{-1} \cdot B_{cl} \cdot r(t), \quad y(t) = C_{cl} \cdot T \cdot z_{cl}(t) \quad (139)$$

with

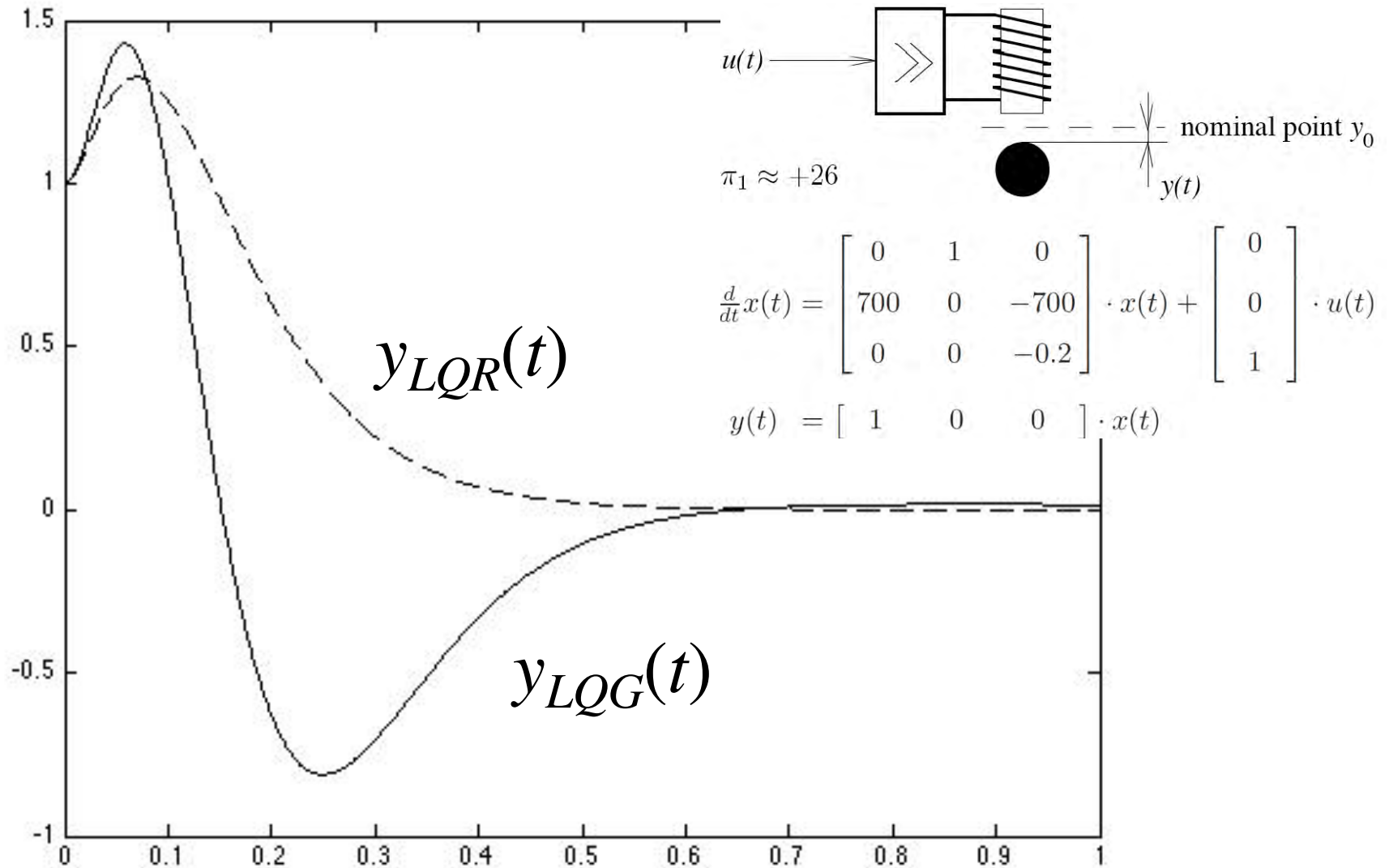
$$T^{-1} \cdot \tilde{A}_{cl} \cdot T = \begin{bmatrix} A - B \cdot K & B \cdot K \\ 0_{n \times n} & A - L \cdot C \end{bmatrix} \quad (140)$$

The Separation Principle immediately follows from the fact that the eigenvalues of block-triangular matrices are equal to the eigenvalues of the triangular blocks and from the fact that the eigenvalues are invariant with respect to coordinate (“similarity”) transformations. Ω

Levitating Sphere – TD Behavior

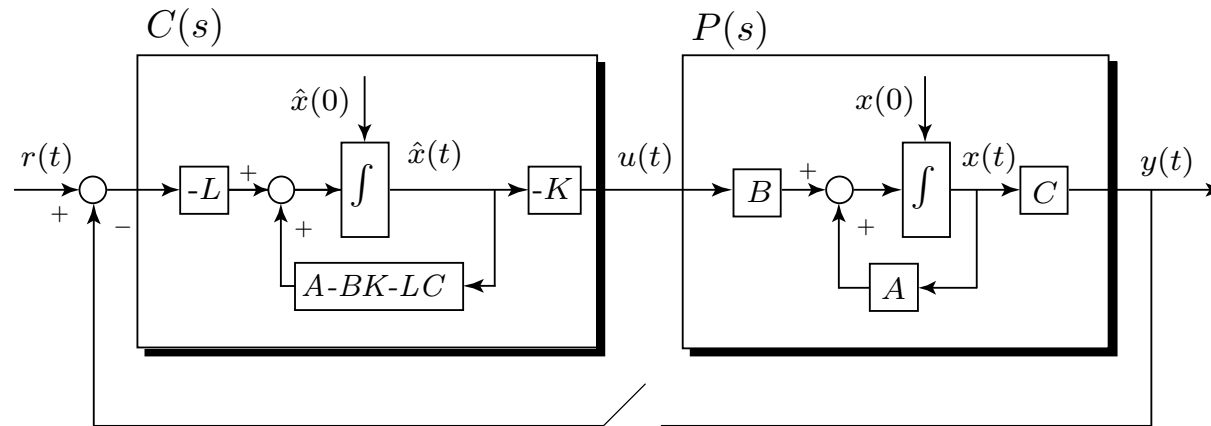
$$x(0) = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

$$\hat{x}(0) = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$$

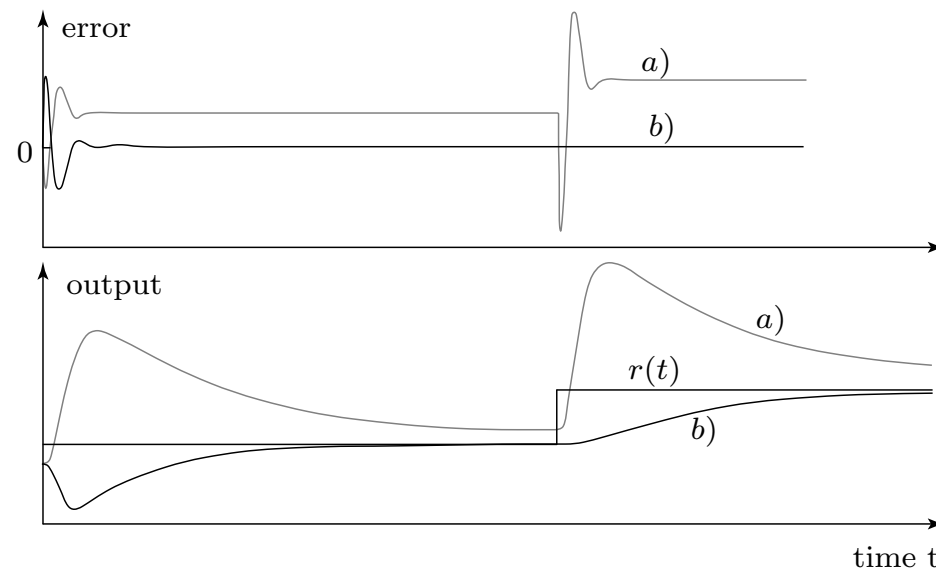


Observer-Based Output Feedback for Reference Tracking

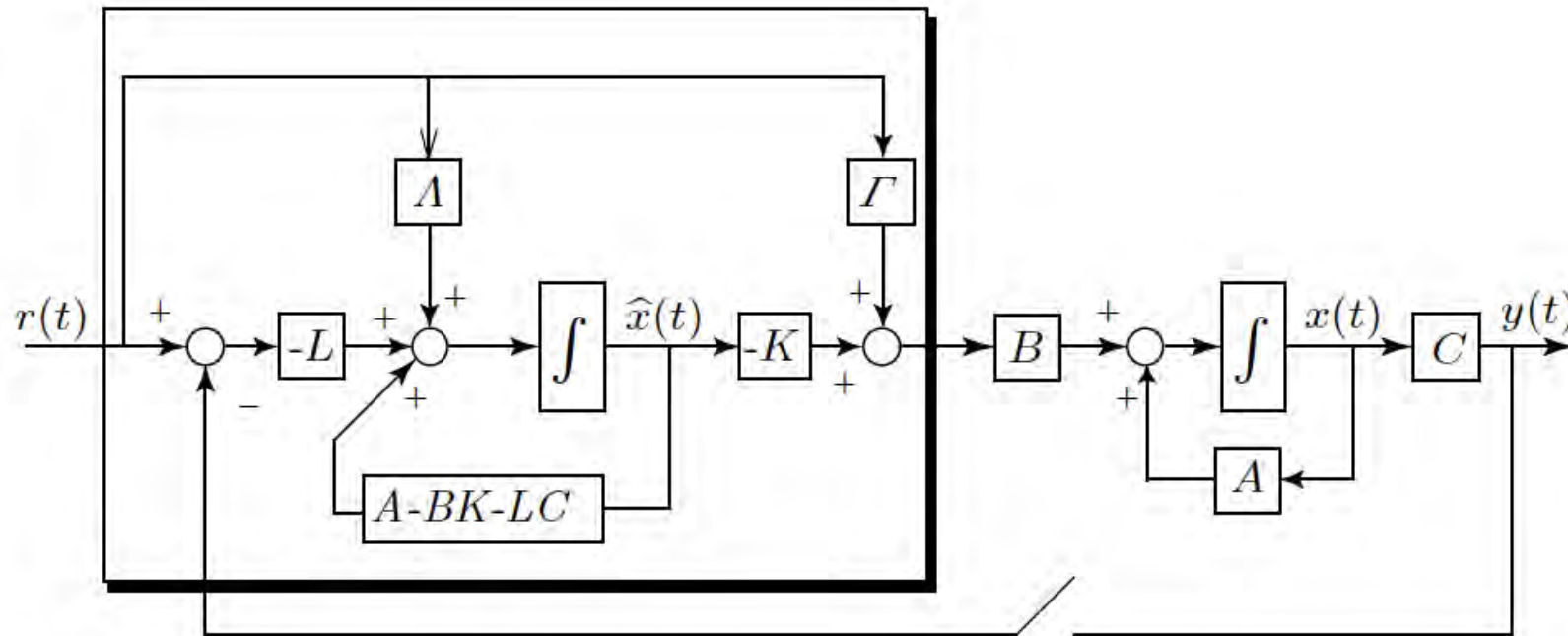
Applying a reference step to



yields curve a)



Unnecessary excitation of error by reference signal may be avoided (curve b) by an appropriate feedforward action (“2-dof controller”)



Choose gains $\Lambda \in \mathbb{R}^{m \times m}$ and $\Gamma \in \mathbb{R}^{m \times m}$ such that:

- the error dynamics do not depend on $r(t)$,
- the static gain $r \rightarrow y$ is equal to I .

Of course more general feedforward systems can be formulated (specifications for all frequencies, not just for $\omega = 0$)!

Solution

$$\Lambda = L + B \cdot \Gamma, \quad \Gamma = - \left(\tilde{C} \cdot \tilde{A}_{\text{cl}}^{-1} \cdot \tilde{B}_r \right)^{-1}$$

where the new matrices define the closed-loop system shown in the last figure, i.e.

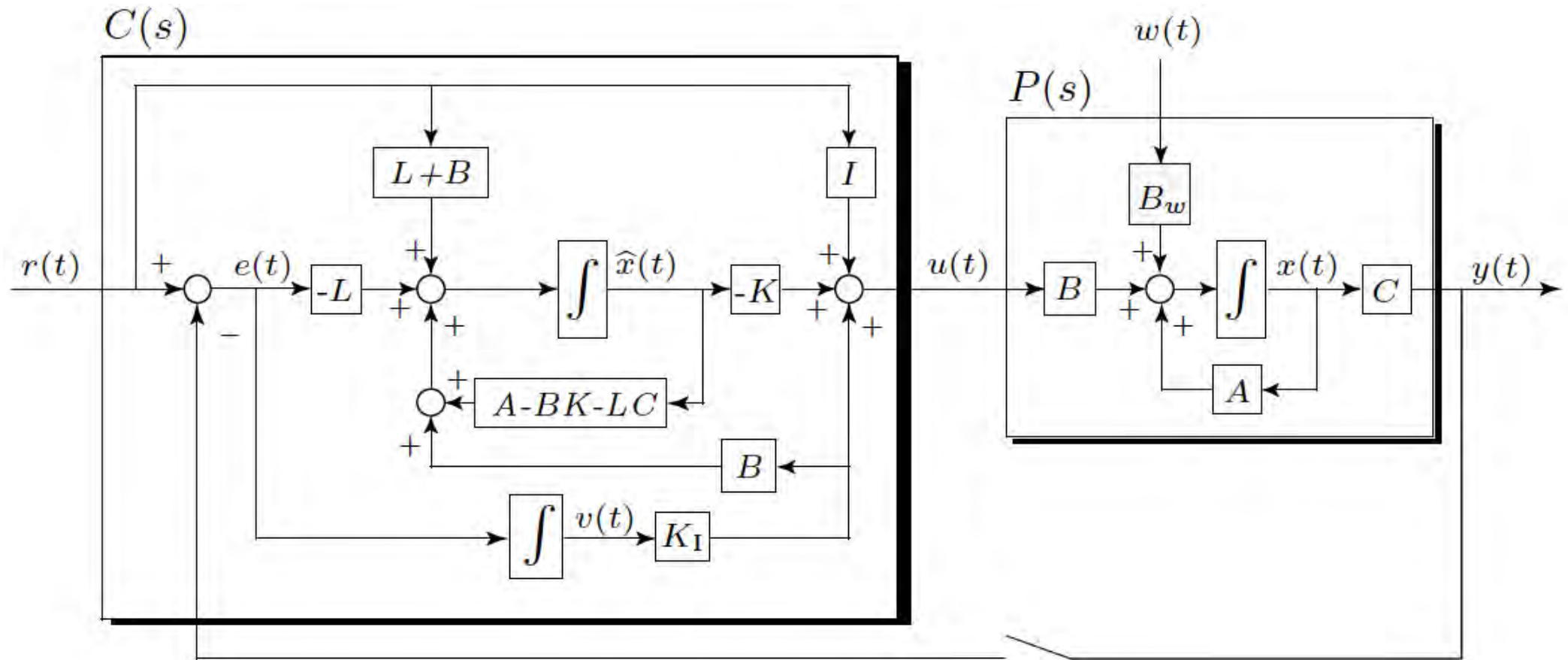
$$\frac{d}{dt} \tilde{x}(t) = \tilde{A}_{\text{cl}} \cdot \tilde{x}(t) + \tilde{B}_r \cdot \Gamma \cdot r(t), \quad y(t) = \tilde{C} \cdot \tilde{x}(t)$$

and

$$\tilde{A}_{\text{cl}} = \begin{bmatrix} A & -B K \\ L C & A - B K - L C \end{bmatrix}, \quad \tilde{B}_r = \begin{bmatrix} B \\ B \end{bmatrix}, \quad \tilde{C} = \begin{bmatrix} C & 0 \end{bmatrix}$$

Notice: Without loss of generality one may assume that $\det \tilde{A} \neq 0$. If the ranks of B and C are full (which is always the case in well-posed problems) then it is also true that $\det\{\tilde{C} \tilde{A}^{-1} \tilde{B}\} \neq 0$.

Output Feedback with Integral Action and Feedforward Part



State-space description open-loop gain $e \rightarrow y$

$$\frac{d}{dt} \begin{bmatrix} x(t) \\ \hat{x}(t) \\ v(t) \end{bmatrix} = \begin{bmatrix} A & -BK & BK_I \\ 0 & A - BK - LC & BK_I \\ 0 & 0 & 0 \end{bmatrix} \cdot \begin{bmatrix} x(t) \\ \hat{x}(t) \\ v(t) \end{bmatrix} + \begin{bmatrix} 0 \\ -L \\ I \end{bmatrix} \cdot e(t)$$

$$y(t) = \begin{bmatrix} C & 0 & 0 \end{bmatrix} \cdot \begin{bmatrix} x(t) \\ \hat{x}(t) \\ v(t) \end{bmatrix}$$

State-space description closed-loop $[r, d]^T \rightarrow y$

$$\frac{d}{dt} \begin{bmatrix} x(t) \\ \hat{x}(t) \\ v(t) \end{bmatrix} = \begin{bmatrix} A & -BK & BK_I \\ LC & A - BK - LC & BK_I \\ -C & 0 & 0 \end{bmatrix} \cdot \begin{bmatrix} x(t) \\ \hat{x}(t) \\ v(t) \end{bmatrix} + \begin{bmatrix} B & B_w \\ B & 0 \\ I & 0 \end{bmatrix} \cdot \begin{bmatrix} r(t) \\ w(t) \end{bmatrix}$$

$$y(t) = \begin{bmatrix} C & 0 & 0 \end{bmatrix} \cdot \begin{bmatrix} x(t) \\ \hat{x}(t) \\ v(t) \end{bmatrix}$$

Design procedure:

1. Form the extended system $\{\tilde{A}, \tilde{B}, \tilde{C}\}$ following the approach outlined in the LQRI section.
2. Design a suitable state-feedback control gain \tilde{K} as a solution of the LQR problem defined by

$$\tilde{K} = [K, -K_I] = \text{lqr}(\tilde{A}, \tilde{B}, \tilde{C}^T \tilde{C}, \rho I)$$

3. Design an observer gain L for the standard system $\{A, B, C\}$ using the duality approach

$$L = \text{lqr}(A^T, C^T, B B^T, \mu I)^T$$

Note: the state v is known and needs not to be included into the observer.

4. The feedforward part is straightforward because the closed-loop feedback system is known to have a DC gain of 1.

